



PSV-2019

Fire Scenario

wetted Surface



PSV SIZING PROCEDURE FOR UNWETTED FIRE SCENARIO

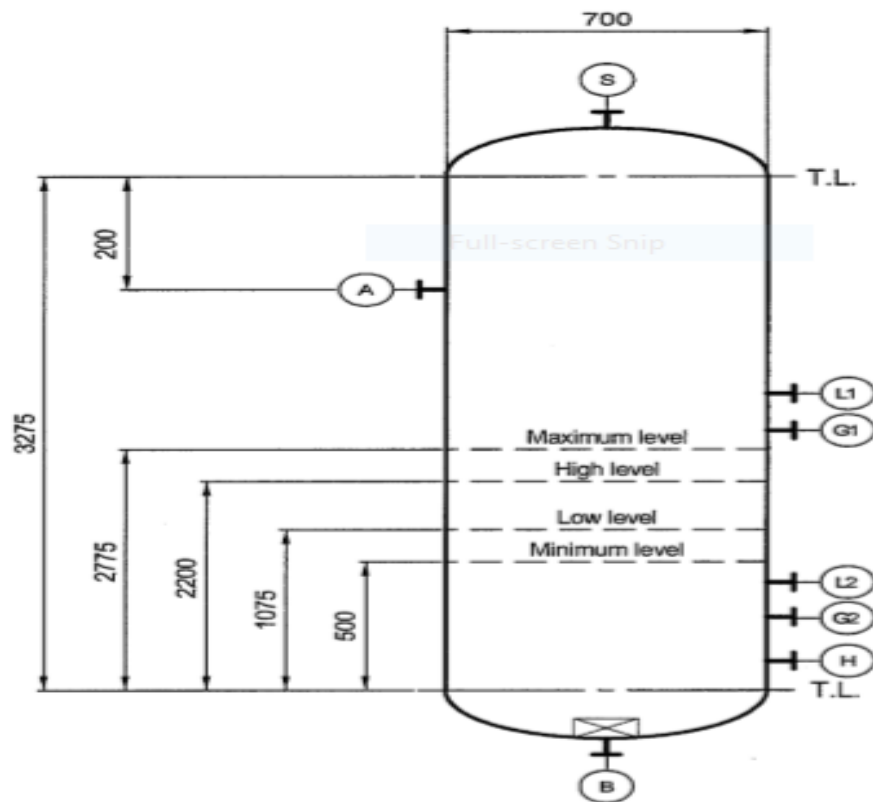
1. Determine the scenario, using API-521
2. Calculate the relief load, using API-520 Part1
3. Calculate the orifice area, using API-520 Part1
4. Select proper PSV type by checking backpressure
5. Use API-526 to determine the designation and the inlet and outlet sizing
6. Use API-520 Part2 to detail its construction



1. Determine the scenario, using API-521

Since it is exposed to fire then a fire scenario is defined.

| Parameters | Value | Parameters | Value |
|------------|-----------------|--------------------|-----------|
| Diameter | 0.7 m | M | 18.02 |
| Height | 3.275 m | Set Pressure | 52 barg |
| Fluid | Steam Condensat | Relieving Pressure | 63.9 bara |
| Z | 0.78 | Accumulation | 0.21 |
| Cp/Cv | 1.09 | Material | CS |





2. Calculate the relief load, using API-520 Part1

$$Q = C_1 \times F \times A_{ws}^{0.82} \quad (7)$$

where

Q is the total heat absorption (input) to the wetted surface, expressed in W (Btu/h);

C_1 is a constant [= 43,200 in SI units (21,000 in USC units)];

F is an environment factor (see Table 5);

A_{ws} is the total wetted surface, expressed in m^2 (ft^2).

NOTE 1 See 4.4.13.2.2 and Table 4.

NOTE 2 The expression, $A_{ws}^{0.82}$, is the area exposure factor or ratio. This ratio recognizes that large vessels are less likely than small ones to be completely exposed to the flame of an open fire.

Where adequate drainage and firefighting equipment do not exist, Equation (8) should be used

$$Q = C_2 \times F \times A_{ws}^{0.82}$$

C_2 is a constant [= 70,900 in SI units (34,500 in USC units)].



Calculation

| Parameters | Value |
|----------------------|---------------------------|
| A_w | 7.73 m² |
| C₂ | 70900 |
| F | 1 |
| λ | 2880 |
| Relief load | 474 kg/h |

3. Calculate orifice area

Determine if it is in critical flow:

$$\frac{P_{cf}}{P_1} = \left[\frac{2}{k+1} \right]^{\frac{k}{k-1}}$$

where

P_{cf} is the critical flow nozzle pressure;

P_1 is the upstream relieving pressure;

k is the ratio of specific heats (C_p/C_v) for an ideal gas at relieving temperature.

If so, then:

$$A = \frac{W}{CK_d K_b K_c} \sqrt{\frac{TZ}{M}}$$



where

- A is the required discharge area of the device, in.² (mm²) (see 5.2);
- W is the required flow through the device, lb/h (kg/h);
- C is a function of the ratio of the ideal gas specific heats ($k = C_p/C_v$) of the gas or vapor at inlet relieving temperature.

The coefficient, C , is determined as follows.

In USC units [for use in Equation (6) through Equation (8) only]:

$$C = 520 \sqrt{k \left(\frac{2}{k+1} \right)^{\frac{(k+1)}{(k-1)}}} \quad (12)$$

In SI units [for use in Equation (9) through Equation (11) only]:

$$C = 0.03948 \sqrt{k \left(\frac{2}{k+1} \right)^{\frac{(k+1)}{(k-1)}}} \quad (13)$$

- K_d is the coefficient of discharge; for preliminary sizing, use the following effective values:
 - 0.975, when a PRV is installed with or without a rupture disk in combination;
 - 0.62, when a PRV is not installed and sizing is for a rupture disk in accordance with 5.12.1.2;
- P_1 is the upstream relieving pressure, psia (kPa); this is the set pressure plus the allowable overpressure (see 5.4) plus atmospheric pressure;
- K_b is the capacity correction factor due to backpressure; this can be obtained from the manufacturer's literature or estimated for preliminary sizing from Figure 31. The backpressure correction factor applies to balanced bellows valves only. For conventional and pilot-operated valves, use a value for K_b equal to 1.0 (see 5.3). See 5.6.4 for conventional valve applications with backpressure of a magnitude that will cause subcritical flow;
- K_c is the combination correction factor for installations with a rupture disk upstream of the PRV (see 5.12.2);
 - equals 1.0 when a rupture disk is not installed;
 - equals 0.9 when a rupture disk is installed in combination with a PRV and the combination does not have a certified value;
- T is the relieving temperature of the inlet gas or vapor, °R(°F + 460) [K(°C + 273)];
- Z is the compressibility factor for the deviation of the actual gas from a perfect gas, evaluated at inlet relieving conditions;
- M is the molecular weight of the gas or vapor at inlet relieving conditions; various handbooks carry tables of molecular weights of materials, but the composition of the flowing gas or vapor is seldom the same as that listed in tables. This value should be obtained from the process data. Table 10 lists values for some common fluids, lbm/lb-mole (kg/kg-mole);
- V is the required flow through the device, SCFM (Nm³/min);
- G_v is the specific gravity of gas at standard conditions referred to air at standard conditions (normal conditions); in other words, $G_v = 1.00$ for air at 14.7 psia and 60 °F (101.325 kPa and 0 °C).



5. Use API-526 to determine the designation and the inlet and outlet sizing

Since it is less than 0.11 inch then D is selected. Also, by checking its rating and temperature limitation, 1D2 is selected. But Topsoe has selected 1E2 in site.

| Designation | Effective Orifice Area (in. ²) |
|-------------|--|
| D | 0.110 |
| E | 0.196 |
| F | 0.307 |
| G | 0.503 |
| H | 0.785 |
| J | 1.287 |
| K | 1.838 |
| L | 2.853 |
| M | 3.60 |
| N | 4.34 |
| P | 6.38 |
| Q | 11.05 |
| R | 16.00 |
| T | 26.00 |

Table 3—Spring-loaded Pressure-relief Valves “D” Orifice ^f (Effective Orifice Area = 0.110 in.²)

| Materials ^b | Valve Size | ASME Flange Class | | Maximum Inlet Flange (Set) Pressure Limit ^a (psig) | | | | | | Outlet Pressure Limit ^a (psig) | | Center-to-Face Dimensions (in.) | |
|---|--------------------|-------------------|------------|---|--------|----------------|--------------|--------------|--------|---|-----------------------------------|---------------------------------|------------|
| | | INLET | OUTLET | Conventional and Balanced Bellows Valves | | | | | | Flange Rating Limit ^a | Bellows Rating Limit ^a | INLET | OUTLET |
| -450 °F to -76 °F | -75 °F to -21 °F | | | -20 °F to 100 °F | 450 °F | 800 °F | 1000 °F | 100 °F | 100 °F | | | | |
| Temperature Range Inclusive -20 °F to 800 °F | | | | | | | | | | | | | |
| Carbon Steel | 1D2 | 150 | 150 | | | 285 | 185 | 80 | | 285 | 230 | 4 1/8 | 4 1/2 |
| | 1D2 ^c | 300 | 150 | | | (285) | (285) | (285) | | 285 | 230 | 4 1/8 | 4 1/2 |
| | 1D2 | 300 | 150 | | | 740 | 620 | 410 | | 285 | 230 | 4 1/8 | 4 1/2 |
| | 1D2 | 600 | 150 | | | 1480 | 1235 | 825 | | 285 | 230 | 4 1/8 | 4 1/2 |
| | 1 1/2D2 | 900 | 300 | | | 2220 | 1855 | 1235 | | (600) | 500 | 4 1/8 | 5 1/2 |
| | 1 1/2D2 1 1/2D3 | 1500 2500 | 300 300 | | | 3705 (6000) | 3090 5150 | 2055 3430 | | (600) 740 | 500 500 | 4 1/8 5 1/2 | 5 1/2 7 |
| Temperature Range Inclusive 801 °F to 1000 °F | | | | | | | | | | | | | |
| Chrome Molybdenum Steel | 1D2 | 300 | 150 | | | | | 510 | 215 | 290 | 230 | 4 1/8 | 4 1/2 |
| | 1D2 | 600 | 150 | | | | | 1015 | 430 | 290 | 230 | 4 1/8 | 4 1/2 |
| | 1 1/2D2 | 900 | 300 | | | | | 1525 | 650 | (600) | 500 | 4 1/8 | 4 1/2 |
| | 1 1/2D2 | 1500 | 300 | | | | | 2540 | 1080 | (600) | 500 | 4 1/8 | 4 1/2 |
| | 1 1/2D3 | 2500 | 300 | | | | | 4230 | 1800 | 750 | 500 | 5 1/2 | 7 |



Select proper PSV type by checking backpressure

According to licensor data, superimposed and build-up backpressure are max 21 barg. Even though selecting Conventional type is not fully recommended, Conventional type has been selected by LESER.

| superimposed | Build-up | Total |
|--------------|----------|---------|
| 0 barg | 21 barg | 21 barg |
| 2.5% | 30% | 40% |

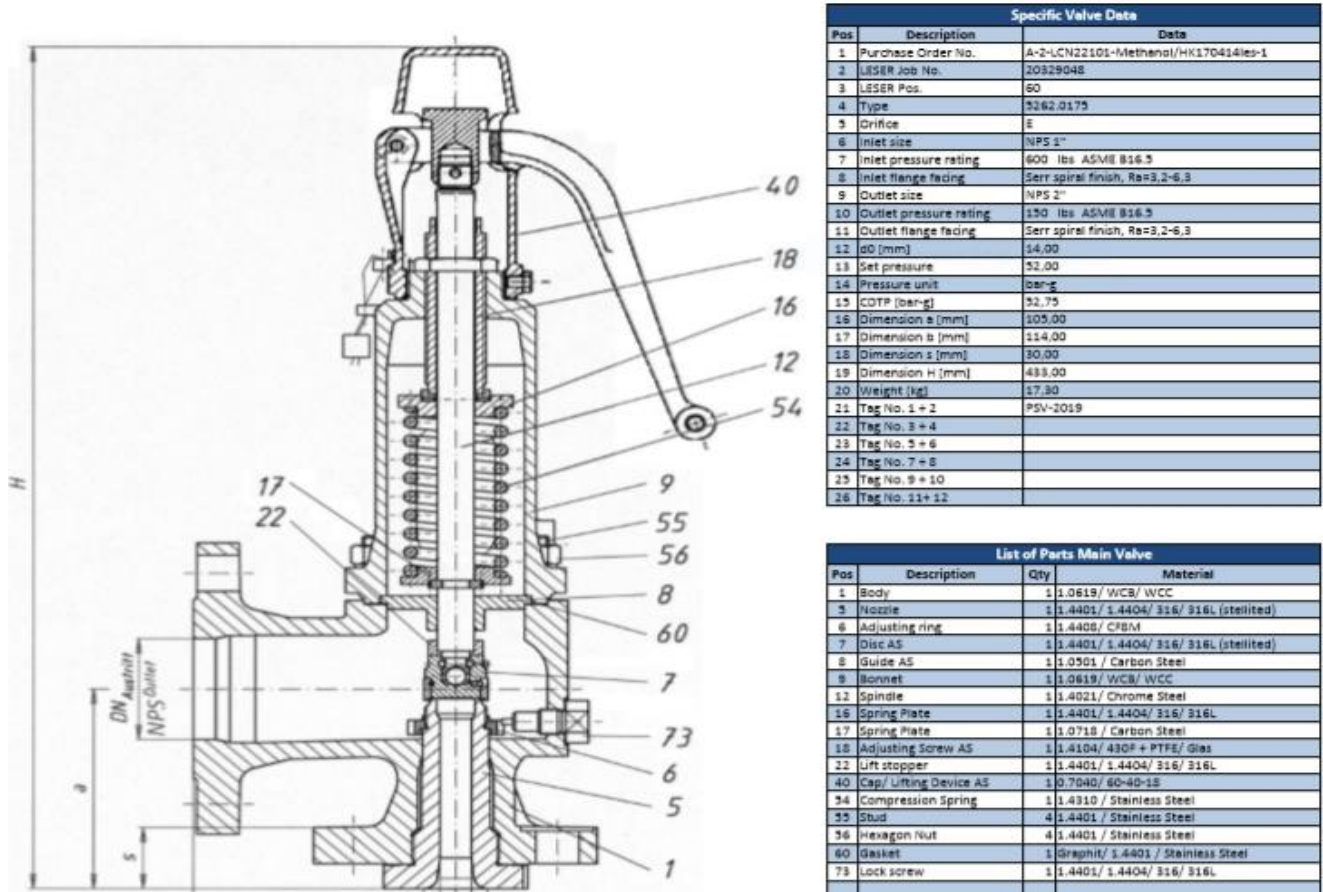
Table 9.1 Maximum backpressure percentages on gas/vapour applications

| Backpressure Type | Effects on valves | | | | Selection |
|-----------------------|----------------------|---|---|------------------------------------|---------------------------------|
| | Value (% of set) | Conventional | Balanced Spring Valve | Pilot Operated | |
| Constant | <30% ¹ | Set point increased by backpressure ³ | No effect | No effect | Conventional, balanced or POSRV |
| | 30%–50% | | Lift/capacity reduced (coefficient) ⁶ | | |
| | >50% ² | Set point increased by backpressure; flow becomes subsonic ⁴ | Generally unstable <i>Do not use</i> | Flow becomes subsonic ⁴ | Conventional or POSRV |
| Variable superimposed | <10% | Set point varies with backpressure ⁵ | No effect | No effect | Balanced or POSRV |
| | 10%–30% ¹ | Unstable | | | |
| | 30%–50% | <i>Do not use</i> | Lift/capacity reduced (coefficient) ⁶ | | |
| | >50% ² | | Generally unstable <i>Do not use</i> | Flow becomes subsonic ⁴ | POSRV only |
| Variable built-up | <10% | No effect | No effect | No effect | Conventional, balanced or POSRV |
| | 10%–30% ¹ | Unstable | | | |
| | 30%–50% | <i>Do not use</i> | Lift/capacity reduced (manufacturer coefficient) ⁶ | | |
| | >50% ² | | Generally unstable <i>Do not use</i> | Flow becomes subsonic ⁴ | POSRV only |



Material Selection

Since it is steam condensate then A-216 WCB is selected for its body



Discussion

1. The relief load calculated by TCC greatly differs by that of TOPSOE and that of mine which stems from λ value in calculation. The λ value for TCC is 1600 kJ/kg while it is 2880 kJ/kg.
2. Another matter is orifice designation dedicated. TOPSOE has selected 1E2 but it appears that 1D2 is also suitable.
3. According to total backpressure calculation, it seems that if balanced type had been selected, it would have more promising performance